Greene & Donoho

Duty Trial of a
Producer-Gas-Engine-Driven
26-Inch Centrifugal Pump

Mechanical Engineering

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# DUTY TRIAL OF A PRODUCER-GAS-ENGINE-DRIVEN 26-INCH CENTRIFUGAL PUMP

BY

# William Bertram Greene Earl Willoughby Donoho

THESIS FOR THE DEGREE OF BACHELOR OF SCIENCE
IN MECHANICAL ENGINEERING

IN THE
COLLEGE OF ENGINEERING
OF THE
UNIVERSITY OF ILLINOIS
PRESENTED JUNE, 1908

#### UNIVERSITY OF ILLINOIS

June 1, 1908

THIS IS TO CERTIFY THAT THE THESIS PREPARED UNDER MY SUPERVISION BY

WILLIAM BERTRAM GREENE EARL WILLOUGHBY DONOHO

ENTITLED DUTY TRIAL OF A PRODUCER-GAS-ENGINE-DRIVEN 26-INCH

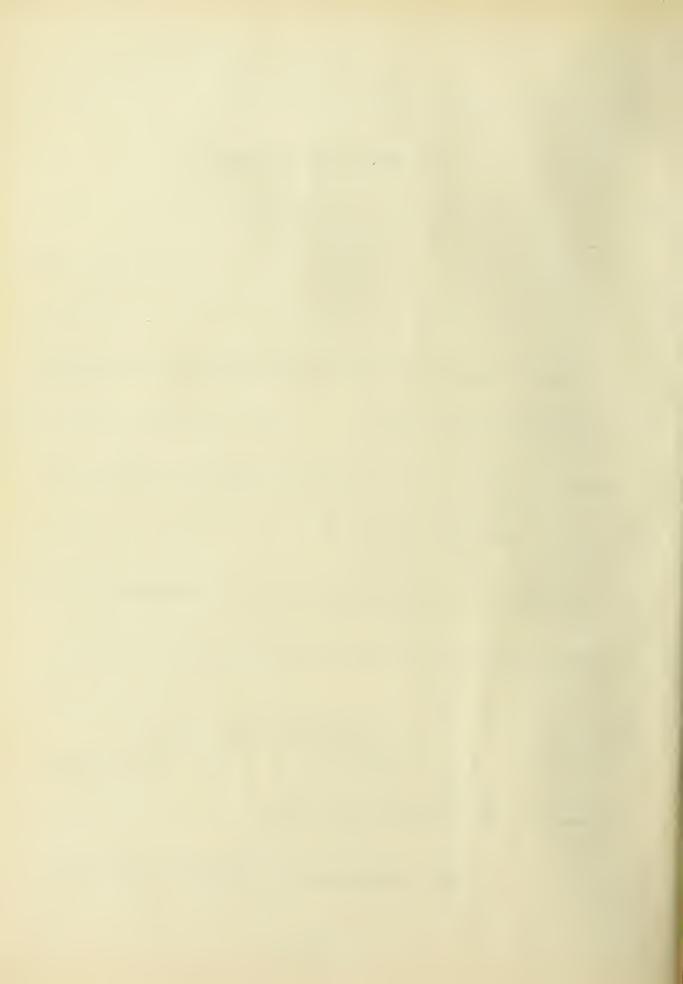
CENTRIFUGAL PUMP

IS APPROVED BY ME AS FULFILLING THIS PART OF THE REQUIREMENTS FOR THE

DEGREE OF Bachelor of Science in Mechanical Engineering

J.O. Tho Jew Propennege

HEAD OF DEPARTMENT OF Mechanical Engineering

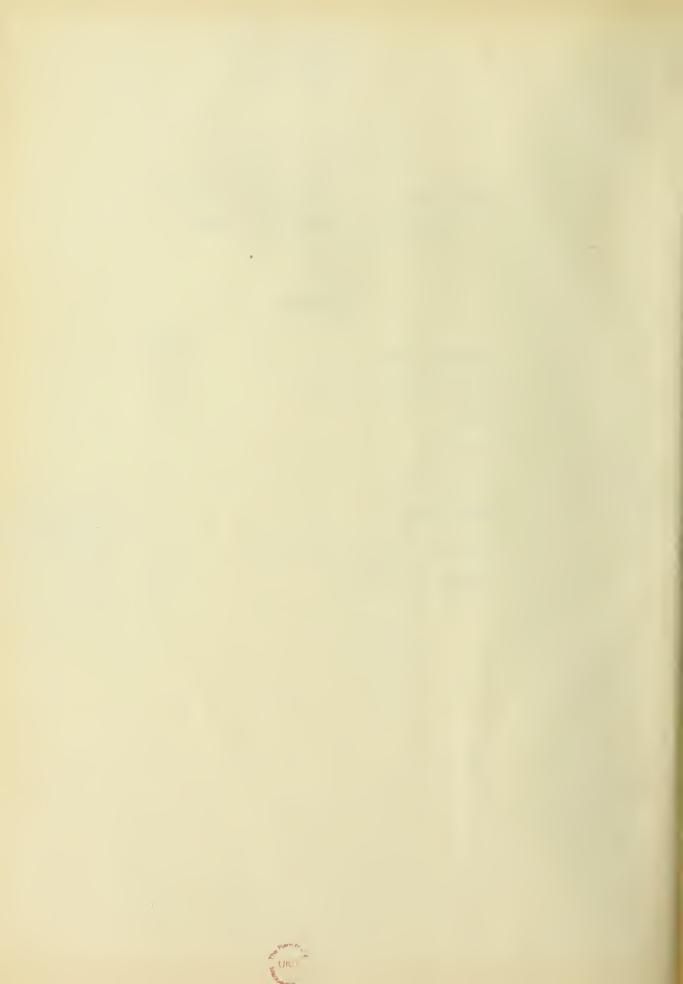


#### OUTLINE OF THESIS. -0-

# DUTY TRIAL OF A PRODUCER GAS ENGINE DRIVEN 26" CENTRIFUGAL PUMP.

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# DUTY TRIAL OF A PRODUCER GAS ENGINE DRIVEN 26" CENTRIFUGAL PUMP.

-0-

#### INTRODUCTION.

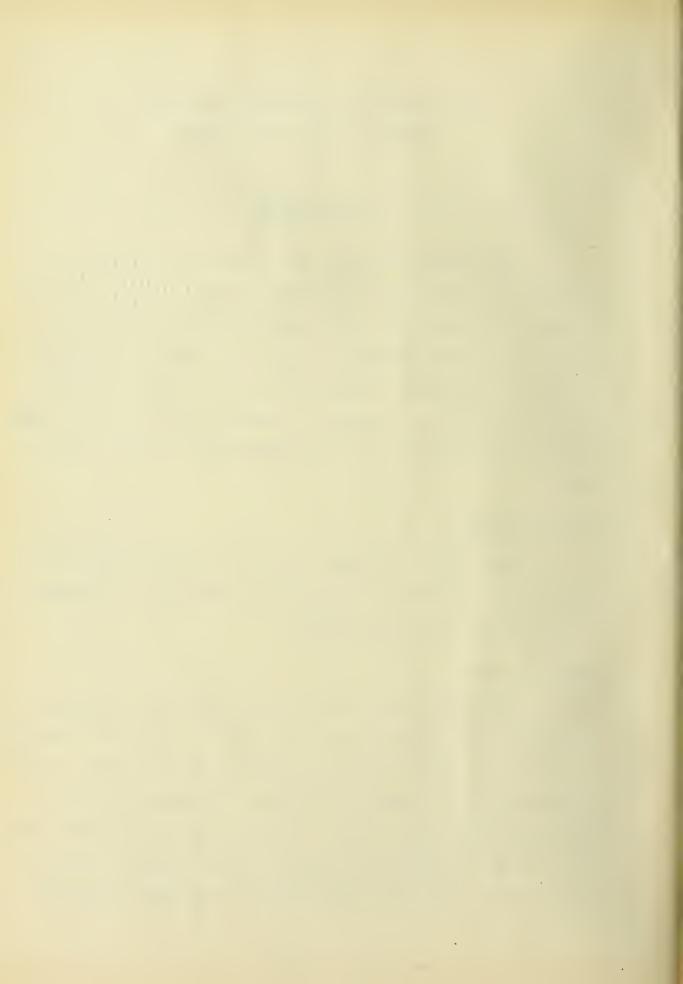
This thesis consists of a commercial test made on a producer gas engine driven centrifugal pump at Pekin, Illinois. The plant is new and the tests herein discussed served in a measure as contract acceptance tests. The plant is owned by the Pekin and La Marge Drainage District, an organization of farmers formed for the purpose of erecting and operating a pump to drain the water from their low farm land into the Illinois river.

#### OBJECT OF TESTS.

The object of these tests, as indicated above is to determine the conformity of the actual efficiency of the pump with the manufacturer's guarantee.

#### NUMBER OF TESTS.

On March 25 and 26 tests were made on the complete plant, each test lasting about eight hours. During the first test the engine stopped once and several times it slowed down, and accordingly the second test was run in hopes of getting better operation of the engine. Between the two tests the engine main bearings and reciprocating parts were carefully examined and adjusted. A one hour test for the mechanical efficiency



of the engine was run on March 27 to permit an analysis of the performance of the component parts of the installation.

#### DESCRIPTION OF PLANT.

PUMP.

The pump tested was a Lawrence Double Suction 26"

Centrifugal Pump, run at 210 R. P. M. It was driven thru a continuous rope drive of 1 1/4" rope of nine turns as shown on drawing, page. The pump was located about two feet above the suction level in a small "lean to" which adjoined the engine house. The engine house floor was seven feet above the pump.

#### ENGINE.

A vertical three cylinder four cycle 14" x 15" Webber engine rated at 125 H.P. was used to run the pump. Ignition was taken care of by a small magneto; governing was done by means of throttling the mixture.

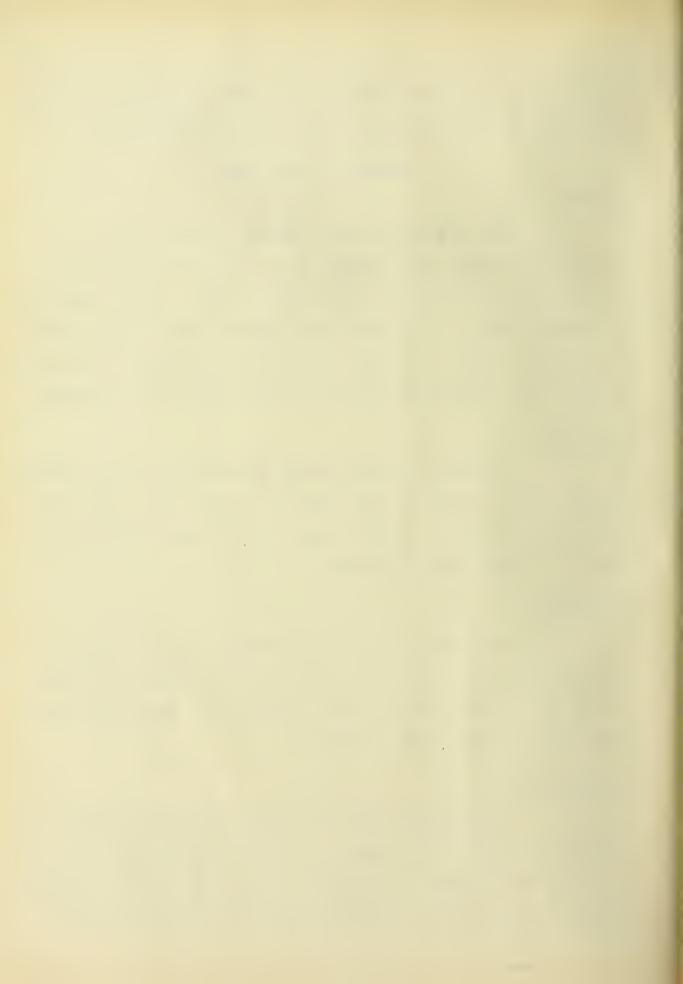
#### PRODUCER.

Two producers had been installed with one scrubber between them; only one of the producers was needed for operation and the same one was used continuously. They were Webber Suction Producers rated at 125 H.P.

#### ACCESSARIES.

A jack shaft over head could either be connected to the large engine or it could be run by a small gasoline engine.

To this were connected a suction fan for starting the producer, a small rotary pump for circulating the cooling water and a



small generator for lighting the plant. An air compressor was connected directly to the gasoline engine and pumped air into a tank; by means of this compressed air the large engine was started.

#### TESTS.

The following observations were made at ten minute intervals thruout the test with exception of "General Data" which was taken every hour.

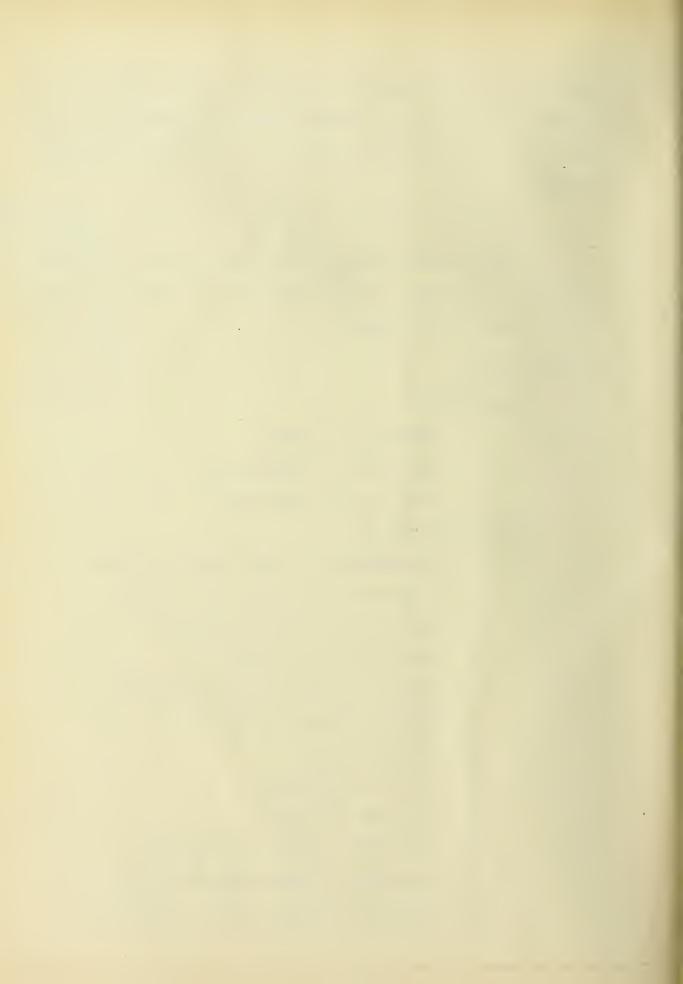
#### 1. DUTY TRIALS.

#### General Data.

- 1. Barometer reading.
- 2. External air temperature.
- 3. Engine room temperature.

## Engine performance.

- 4. Temperature of jacket water, initial.
- 5. Temperature of jacket water final.
- 6. Temperature of gas to engine.
- 7. Temperature of exhaust.
- 8. Weight of jacket water per minute.
- 9. Weight of coal to producer.
- 10. Weight of ash.
- 11. R. P. M. of Engine.
- 12. F. P. M. of engine.
- 13. Indicator diagram, Cylinder No. 1.
- 14. Indicator diagram, Cylinder No. 2.
- 15. Indicator diagram, Cylinder No. 3.



#### Pump Performance.

- 16. Temperature of Suction Water.
- 17. Suction head.
- 18. Delivery head.
- 19. Velocity head in suction pipe, Pitot tube reading.
- 20. Velocity head in discharge pipe, Pitot tube reading.
- 21. R. P. M. of pump.

#### 2. MECHANICAL EFFICIENCY TEST.

#### General Data.

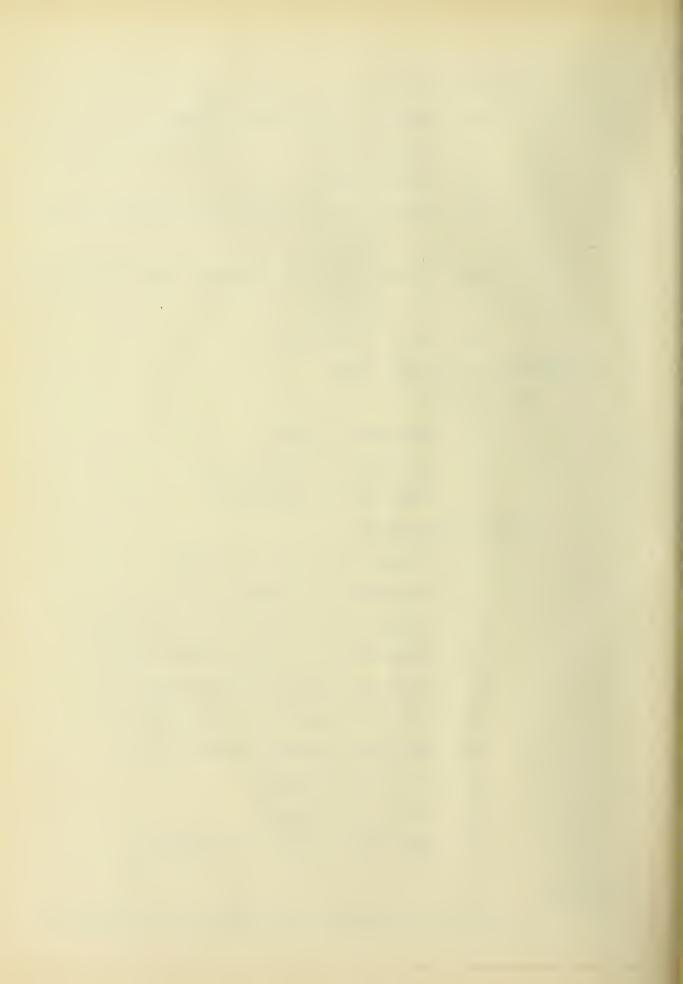
- 1. Barometer reading.
- 2. External air temperature.
- 3. Engine room temperature.

## Engine Performance.

- 4. Temperature of gas to engine.
- 5. Temperature of exhaust gas.
- 6. Temperature of jacket water, initial.
- 7. Temperature of jacket water, final.
- 8. Indicator diagram, Cylinder No. 1.
- 9. Indicator diagram, Cylinder No. 2.
- 10. Indicator diagram, Cylinder No. 3.
- 11. R. P. M. of engine.
- 12. E. P. M. of engine.
- 13. Net Weight of brake on scales.

#### METHOD.

In order to determine the mechanical efficiency of



the pump the combined efficiency of the engine and pump had first to be obtained by taking the ratio of the water horse power to the indicated horse power. This ratio divided by the mechanical efficiency of the engine gives the pump efficiency including transmission.

#### TO GET PLANT EFFICIENCY.

Samples of coal and ash were taken for testing and all coal put into hopper and all ash from the grates was weighed for the purpose of getting the thermal efficiency of the plant. The thermal efficiency is the ratio of the delivered horse-power to the horse power equivalent to the B. T. U.delivered in the coal to the producer, correction being made for heat in refuse.

#### TEMPERATURES.

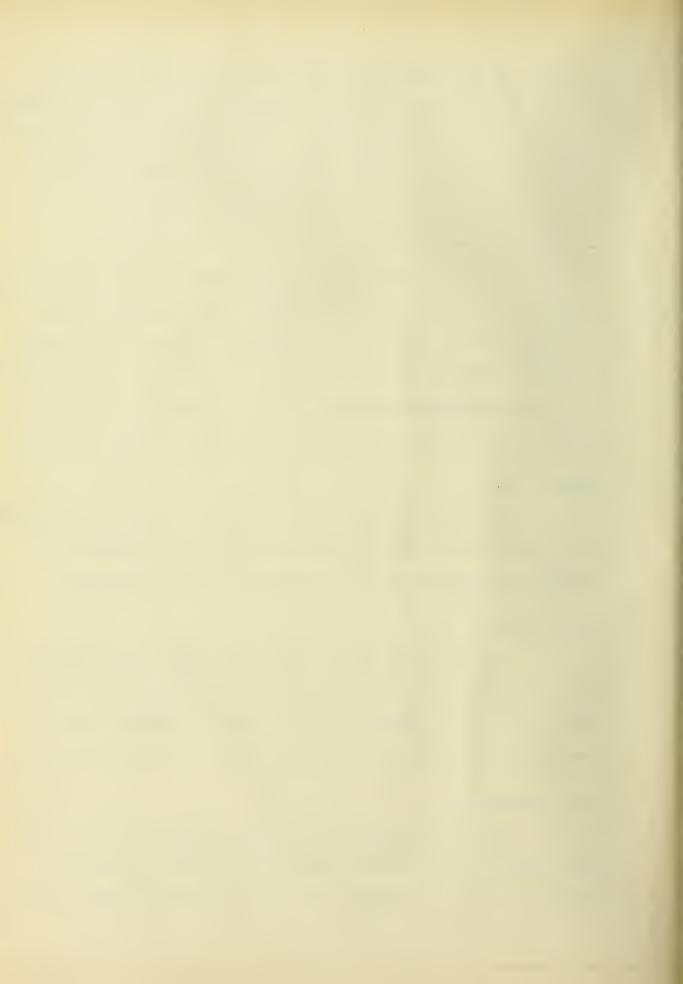
All temperatures of gas, water and air were taken merely for purposes of general information as a heat balance or analysis was impossible in the absence of a gas analysis.

#### JACKET WATER.

The weight of jacket water per minute was obtained by timing the flow into a bucket with a stop watch and weighing the same. This did not enter into the final calculations for the reasons already given of the impractibility of a heat balance.

#### INDICATOR CARDS.

Indicator springs were calibrated after the test was made. The reducing motion consisted of a long cord passing across the top of the engine where the indicators were located,



and guided by pulleys down to the end of the shaft, where there was an arrangement that enabled an operator to hook the cord onto anyone of three cranks which worked in phase with the eifferent cylinders. The accompanying sketch, page , will better show the arrangement.

#### R.P.M. OF ENGINE.

This was obtained by means of a tachometer which had been calibrated with an ordinary speed counter.

#### R.P.M. OF PUMP.

This was obtained by multiplying the engine R.P.M. by the velocity ratio of the two pulleys.

#### VELOCITY HEAD.

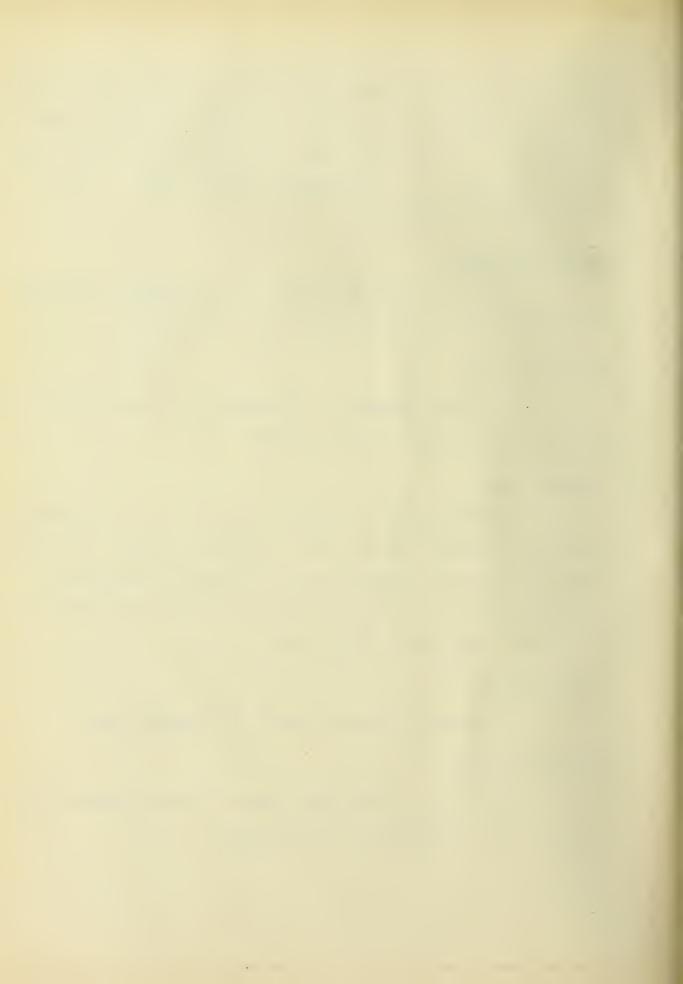
Calibrated Pitot tubes were placed in the discharge and in the suction pipe and were read simultaneously. The reading of the one in discharge pipe was accepted as standard as it was placed in a straight section of pipe and the other had necessarily been placed near a bend.

#### SUCTION HEAD.

This was obtained by means of a vacuum gage.

#### DISCHARGE HEAD.

This was measured with a mercury column connected by tubing to the discharge end of the pump.



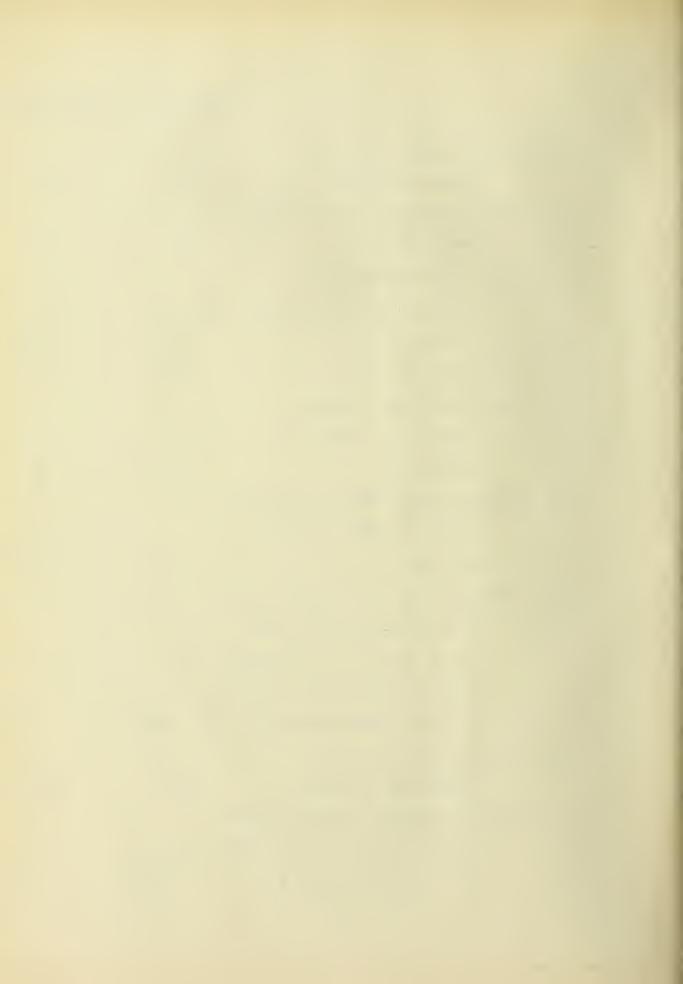
#### RESULTS.

From the above observations and from the coal and ash analyses the following results were calculated.

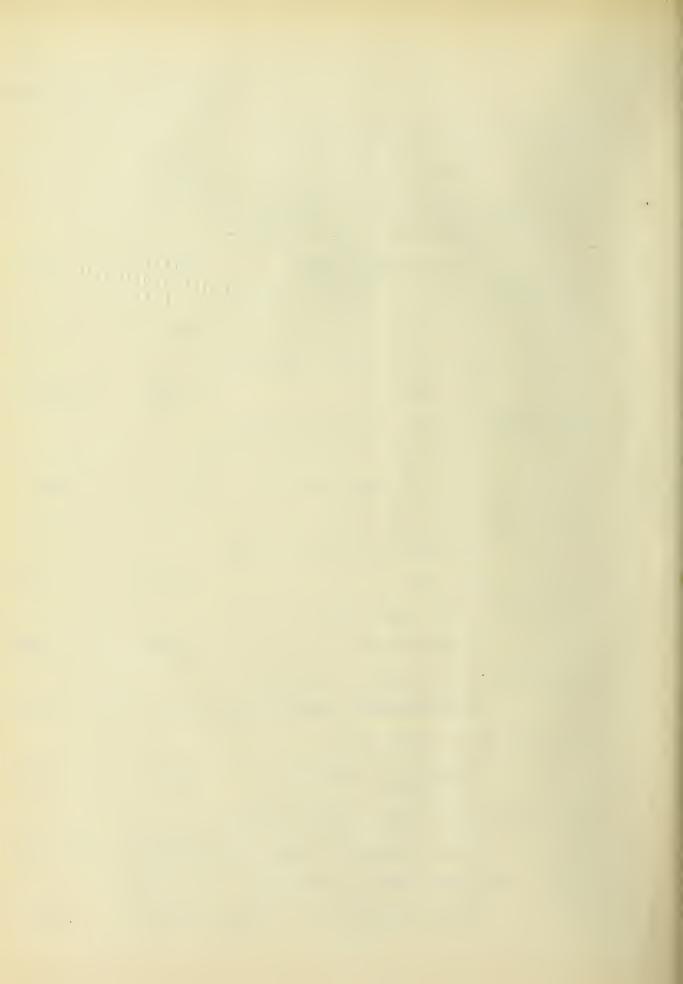
	Item.	Trial No. 1. Mar. 25, '08	Trial No. 2. Mar. 26, '08
1.	Duration of trial, hours,	7.4	7.58
2.	Atmospheric pressure, lbs.		
	per sq. in. (Read-		
	ing No. 1 x .493	14.0	14.15
3.	Average external air		
	temperature (From read-		
	ing No. 2)	66.5°F.	65.2°F.
4.	Average engine room tempera-		
	ture. (From reading No.3)	70.6°F.	70.1°F.
5.	Average temperature of gas to		
	Engine. (From reading		
	No. 6)	68.7°F.	72.3°F.
6.	Average temperature exhaust		
	gases. (From Reading No.7)	208.4°F.	254.6°F.
7.	Average initial temperature of		
	jacket water. (From reading	g	
	No. 4)	54.0°F.	64.3°F.
8.	Average final temperature of		
	jacket water (From reading		
	No. 5)	100.5°F.	141.8°F.
9.	Average temperature difference.		
	("Item 8" - "Item 7")	46.5°F.	77.5°F.



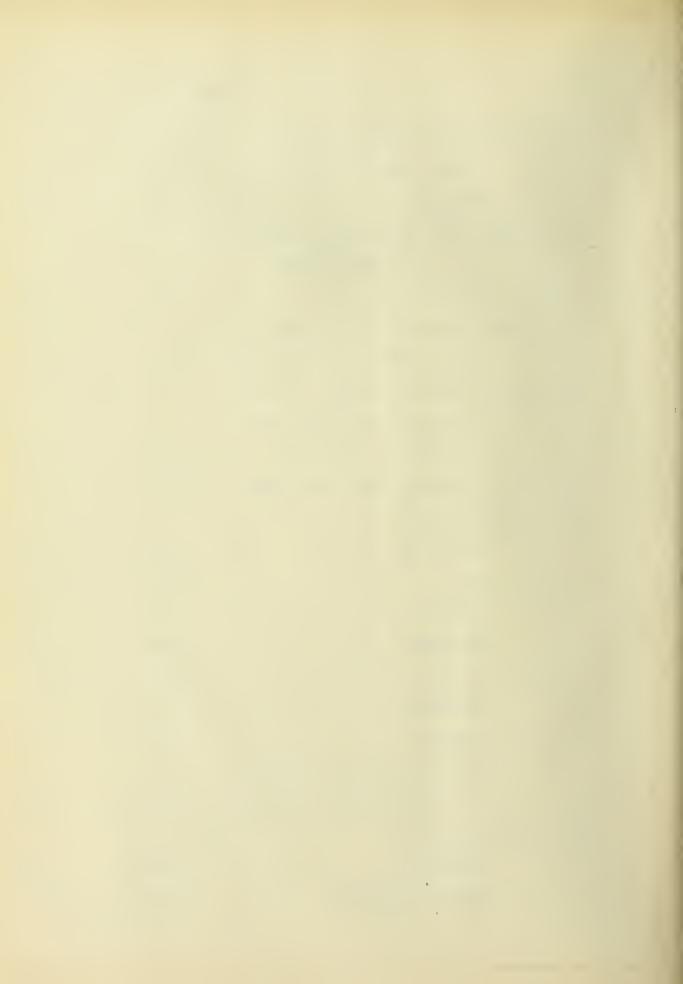
10.	Average R.P.M. of engine		
	(From Reading No. 11)	284	295
11.	Average Explosion per minute.		
	("Item 10" ÷ 2)	142	147.5
12.	I.H.P. cylinder No. 1.		
	(.00584 x M.E.P. x "Item 11")	18.1	27
13.	I.H.P. cylinder No. 2.	17.5	21.9
14.	I.H.P. cylinder No. 3.	25.6	24.1
15.	Total I.H.P. for engine.		
	("Item 12" "Item 13"		
	"Item 14")	61.2	73
16.	Average temperatures of suction		
	water, T. (From reading		
	No. 16)	51.80	59.40
17.	Average velocity of flow in suc-		
	tion pipe in ft. per sec. V'		
	(V' = 2 g h ; h, velocity		
	head in pitot tube, in feet		
	of water from reading No. 19)	8.04	7.96
18.	Average R.P.M. of pump.		
	("Item 10" $\times 54/40$ )	210	219
19.	Average velocity of flow in dis-		
	charge pipe in ft. per sec,		
	V. V = .975 2 g h)	7.53	7.56



20.	Average celocity. (("Item 17"
	"Item 19") ÷ 2) 8.09 7.76
21.	Water pumped, cubic feet per
	second. Q. (Q = V x A, A =
	area discharge pipe) 27.7 27.8
22.	water pumped, gallons per minute,  G.: (G = t0 x 0 x D. D = gallons
	G.; (G = t0 x Q x D, D = gallons
	per cubic foot at temperature
	T, "Item 16") 12,400 12,450
23.	Water pumped, pounds per second.
	("Item 19" x weight per cubic
	foot at temperature, T) 1732 1739
24.	Total gage head, feet.
	(Average reading No. 17 and
	No. 18) 9.06 8.43
25.	Total level head, feet.
	(Measurement) 8.08 8.08
26.	Water horse power. ("Item 23"
	x1 (Reidingdl by 5Reading 18)) 28.6 27.61
27.	Effective horse power.
	"Item 27" x "Item 25" 26.4 26.5
28.	Capacity of pump, gallons per 24
	hours. "Item 22" x 60 x 24. 17840000 17910000
29.	Total coal burned per hour as
	fired. Reading No.9 : Item 1. 117.2 93.2

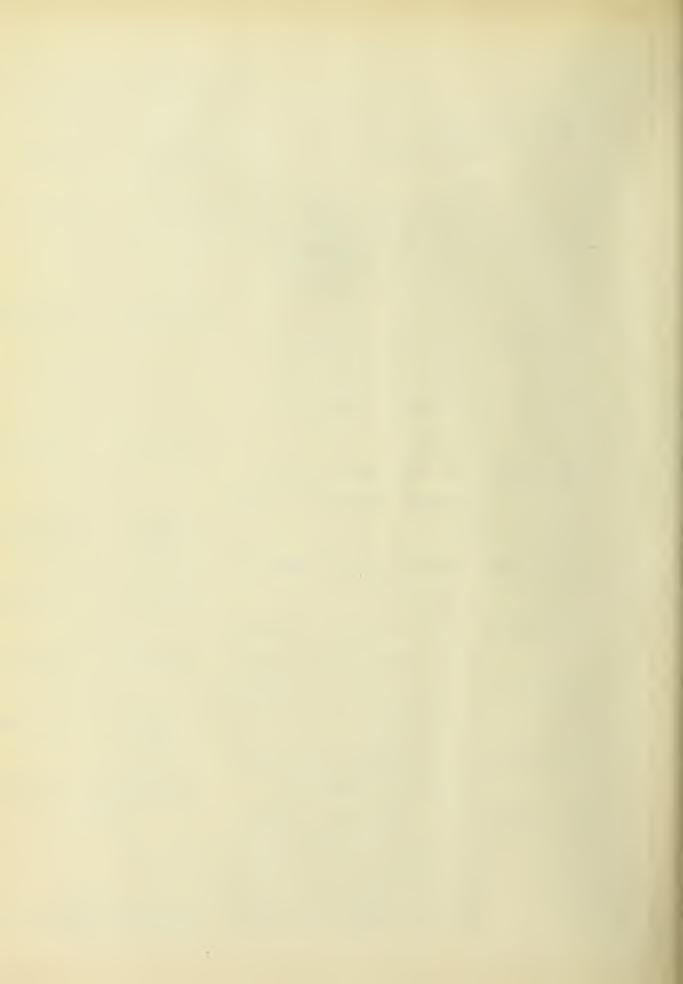


30.	Total coal burned, lbs per		
	I.H.P. hour. "Item 29 :		
	"Item 15"	1.92	1.275
31.	Total coal burned, lbs. per B.H.P.		
	hr. "Item 30" : .786,		
	Mech. eff.	2.44	1.623
32.	Total coal burned, lbs. per I.H.P.		
	hour. "Item 29": "Item 26"	4.1	3.39
33.	Total coal burned, lbs. per ef-		
	fective horse power hour.		
	"Item 29" + "Item 27"	4.44	3.52
34.	Total ash and refuse per hour,	35.20	21.25
35.	Coal analysis.		
	Moisture,	2.15%	1.43%
	Ash	15.83	15.25
	Sulphur	.94	.91
	Hydrogen.	2.80	2.84
	Carbon	75.68	76.76
	Nitrogen	.85	.85
	0xygen	1.75	1.96
	B.T.U. per lb. of coal as		
	fired (Parr calorimeter)	12,330	12,569
36.	Ash analysis.		
	Moisture	1.10%	.64%
	Volitile matter	45.64	42.54



			J. 540 W
		Trial No. 1.	Trial No. 2.
36.	(Continued)		
	Fixed carbon	53.26	56.82
	Ash		
	B.T.U. (calculated)	6861	6417
37.	B.T.U. per hour supplied to		
	the producer. "Item 29 x		
	"Item 35"	1,445,000	1,171,500
38.	B.T.U. per hour supplied to		
	producer and rejected		
	in refuse. "Item 34" x		
	"Item 36"	241,500	136,350
39.	Net B.T.U. evolved per hour in		
	producer. "Item 37" - "It		
	38 <sup>n</sup>	1,203,500	1,035,150
40.	B.T.U. per I.H.P. hr. "Item 39		
	: "Item 15"	23,600	16,045
41.	B.T.U. per B.H.P. hr. "Item 3		20.470
4.0	: (.786, Mech. eff.x I.H.P		20,410
42.	B.T.U. per W.H.P. hr. "Item'3		49.000
A 77	: "Item 26"	48,850	42,820
40.	B.T.U. per; effective H.P. hr.	54,800	45,400
4.1	"Item 39" : "Item 27"  Pump Duty. (Foot-lbs. of work	•	40,400
-EE •	done per 1,000,000 B.T.U.		
	supplied bases upon coal	28	
	-suppried bases upon coar	CLD	

fired and pump horse-power. 39,220,000 44,750,000



45. Overall pumping efficiency.

(Ratio of heat equivalent of W.H.P. to "Item 39")

46. Overall plant efficiency. (Ratio of heat equivalent of effec-

tive horse power to "Item 39) 4.66% 5.75%

47. Potential efficiency of Pro-

ducer, transmission, engine

and pump. "Item 39" and . . .

W.H.P. 6.05% 6.72%

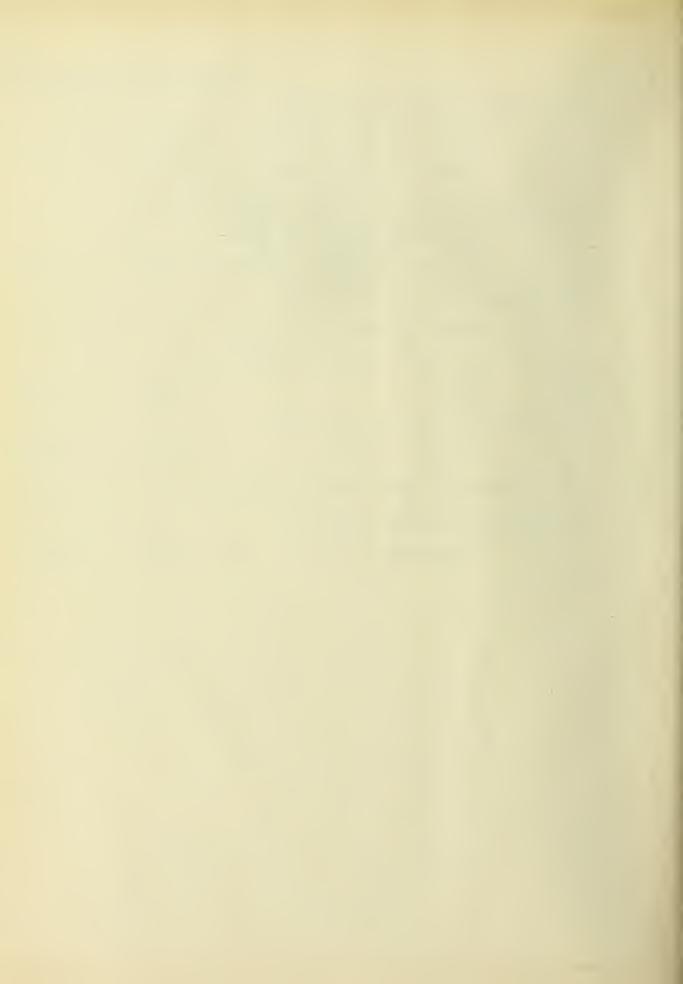
48. Efficiency ratio, Mech. eff.

(B.H.P. ÷ I.H.P.) 78.6% 78.6%

49. Efficiency ratio. (W.H.P. : I.H.P) 46.7% 36.6%

50. Efficiency ratio. (Effective

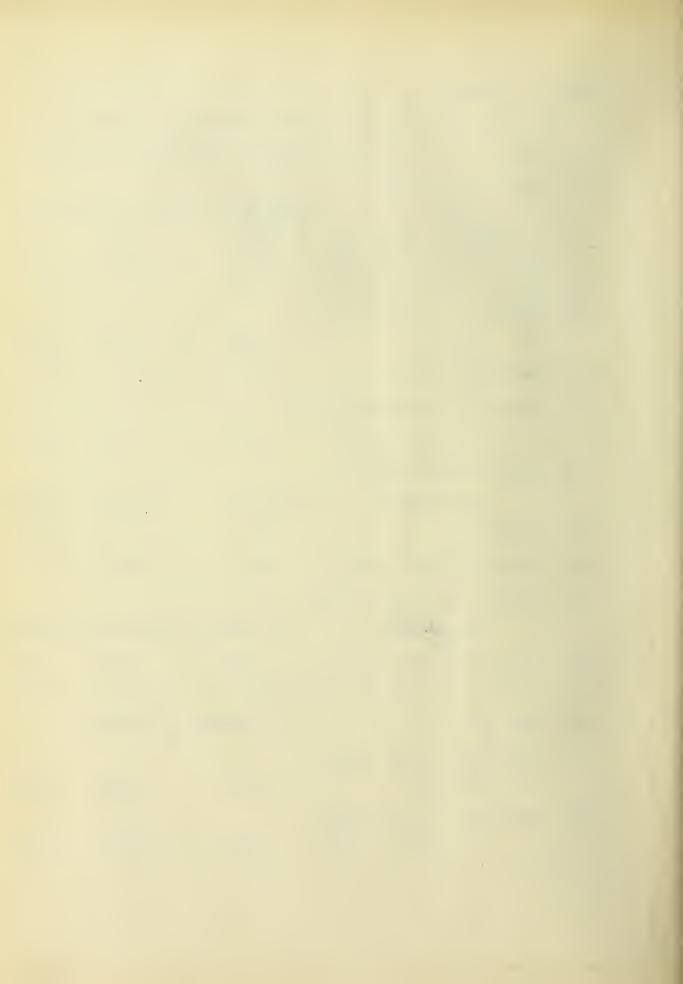
horse power : I.H.P.) 43.3% 36.3%



### SUMMARY OF RESULTS.

The following table shows significant results as taken from the previous table, compared with normal results which should be expected.

	Item.	Trial No.1.	Frial No.2.	Normal.
1.	Duration of trial, hrs.	7.8	7.28	
15.	Total average I.H.P.	61.2	73	147
48.	Mechanical efficiency of	f		
	engine.	78.6%	78.6%	85%
	Estimated B.H.P.	48.1	57.3	125
17.	Velocity in discharge p	ipe,		
	feet per second.	8.64	7.96	10
22.	Water pumped, gallons			
	per minute.	12,400	12,450	16,650
26.	W.H.P.	28.6	26.61	35.4
27.	Effective horse power	26.4	26.50	34
28.	Capacity of pump, gallon	ns		
	per 24 hrs.	17,8740,000	17,910,000	24000000
31.	Coal burned per B.H.P:	2.44	1.62	1.25
35.	B.T.U. per pound of coal	1. 12,330	12,569	12,500
44.	Pump Duty.	39,2200,000	46,580,000	
49.	Efficiency ratio, W.H.P	•		
	÷ I.H.P.	46.7%	36.6%	80%
5 <b>0.</b>	Efficiency ratio. Effec	made .		
	tive H.P. : I.H.P	. 43.3%	36.37%	75%



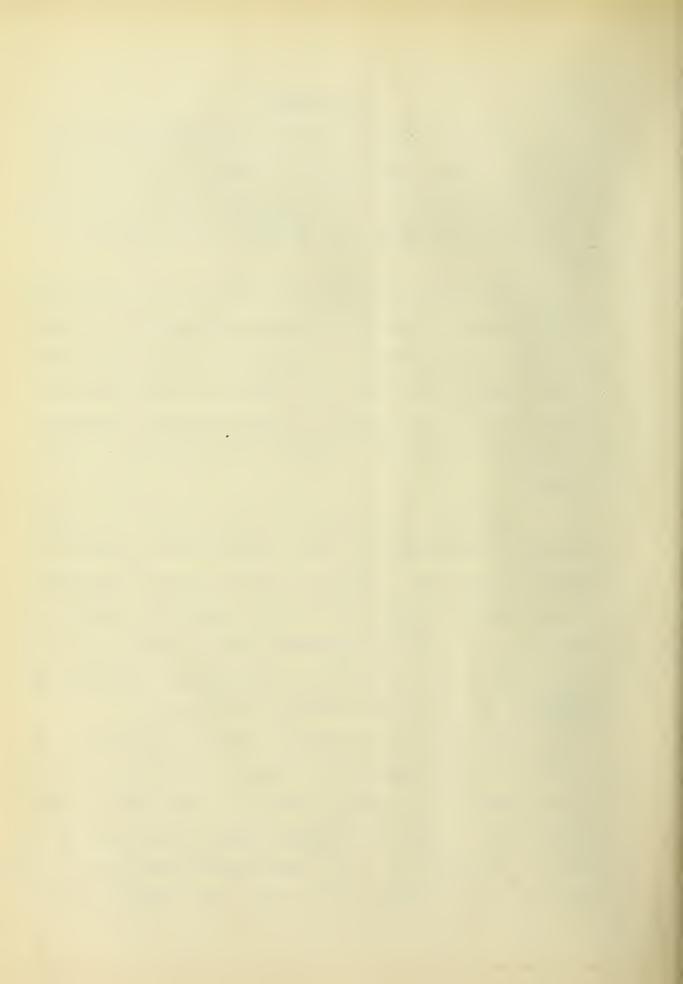
## CONCLUSIONS.

Owing to rather unsatisfactory performance of the engine and producer during test, the results as a whole probably err as high as 5%. Still with the results obtained and the record of performance the following conclusions may be drawn.

The high percentage of unburned carbon in the ash and the large amount of ash and refuse rejected from the producer are a result of too frequent cleaning and poking of the fire and show inefficient firing. The behavior of the engine is accounted for by a poor quality of gas irregularly supplied. The amount of coal burned per B.H.P. hour, 1.62 lb., is much too high.

The load on the engine is less than half of normal rating thus decreasing to a great extent both the thermal and mechanical efficiency. During the brake test for the determination of mechanical efficiency of the engine the indicator on Cylinder No. 2 was out of commission and the total I.H.P. had to be estimated from the previous performance. The mechanical efficiency of the engine as obtained was 78.6%.

The pump was not worked to its full capacity due to the fact that the screen over the end of the intake pipe shut off one fourth of the opening; this also often became clogged; reducing still further the effective area. This condition impairs the efficiency to a very considerable extent and at normal load reduces the capacity of the pump. The velocity of water



in the pipe 7.85 feet per second was lower than normal capacity demands.

The column of normal results to be expected, shown under "Summary of Results" shows a wide variation from those actually obtained.



E EPAINAGE DIST

	1	n Feet	10
139			A of
64.1	1.13	1.00	1.065
67.4	1.13	.96	1.045
66.3	1.13	.93	1.03
65.5	.905	.96	.932
62.8	.905	.98	.942
62.5	.905		
55.0	1.13		
575	.905		
56.0			
62.0	1.13	100	
68.5		.71	
67.0		.85	
68.6			
68.0		.83	
65.0		.96	
		.98	
61.4		.96	
59.5		1.00	
54.7		1.02	
41.4		.67	
55.6		.79	
51.7		.77	
48.4		.77	
44.1		.54	
60.0	1.13	.67	.90
60.1	1.13	.67	.90
60.3	1.35	1.04	1.195

Mechanical Efficency 284 of Pump Cleor & windy

Thours. 24 min. March 25, 38

TEMP.		F		
		TIDS-1	*	
	t t	In Feet		
P.M.	3	610	1	
	300 222 150 20.4 22.0 31.1 24.5 17.9 18.9 27.3	64.1 1.13 1.00 1.065 4.52 510 9.62 8.08 27.9	12,530 753,000 18,080,000 30.6 25.6 .84 .610 R	Patio Engine Puller Dia to Pump
3 55 69 5 72 55 101 46 68 207 540	300 222 150 23.2 19.6 34.0 85.6 20.4 17.2 29.8			Pulley Dia. 234
	300 222 /50 22.7 /90 33.9 25.2 20.0 /66 29.7		12,130 730,000 17,520,000 28.8 24.8 .86 .556	2
	292 216 146 23.6 19.4 33.7 25.6 20.2 16.5 28.8		12,280 737,000 17,700,000 28.0 25.0 .894 .566	
			12,920 776,000 18,600,000 28.5 26.4 .922 .584	
		62.5 .905 3.96 4.75 8.71 8.08	922	
			.903	
	262   94   31   21.6   20.5   29.0   23.9   16.5   16.3   22.2		.927	
7. 12 7.	280 207 140 21.1 19.8 29.4 23.4 173 16.2 24.0	57.5 .905 3.96 4.75 8.71 8.08	.947	~ ~ ~
5.00 69.5 71.5 55 96 41 70 214 54.0	270 200 135 212 190 33.9 24.7 16.9 15.0 24.1	5-6.0 3.68 4.71 8.39 8.08	and the second s	seems of the seems
and the second of the second o	292 216 146 23.2 19.0 30.5 24.2 19.8 16.2 26.0	3.96 4.65 8.61 8.08	939	
	192 216 146 25.3 22.0 33.0 26.7 21.6 18.7 28.2		10,550 635,000 15,200,000 24.0 21.5 .899 .447	
30 54 109 55 70 194 540	295 218 147.5 25.6 19.7 32.4 25.9 22.1 169 28.0		11,550 694,000 16,600,000 26.4 23.6 .899 .576	
30 54 110 56 71 236 54.0	298 221 149 23.9 21.5 33.4 26.3 20.8 18.7 29.1	68.6 4.53 4.65 9.18 8.08		
	300 222 150 26.0 19.8 31.8 25.9 22.8 17.3 27.9		11,450 688,000 16,500,000 26.6 23.4 .88 502	
	292 216 146 244 20.3 31.4 25.2 20.9 17.3 26.8		12,280 738,000 17,700,000 28.0 25.0 .894 .559	
	290 215 145 24.4 20.1 31.4 25.3 20.7 17.0	98 4.53 4.65 9.18 8.08 17.7	12,400 745,000 17,900,000 28.7 25.3 .88	
	286 218 143 22.7 20.4 30.4 24.5 19.0 17.0 25.4	61.4 .96 4.53 4.52 9.05 8.08 17.3	12,180 738,000 17,700,000 28.0 25.0 .894 .583	Generator thrown in.
			12,530 753,000 18,080,000 28.2 25.6 .908 .605	
	280 207 140 20.0 18.9 16.4 15.4 22.9	54.7 1.02 4.06 4.56 8.62 8.08 28.2	12,670 760,000 18,150,000 27.6 25.9 .938 .645	
	260 193 130 19.6 17.0 28.0 21.5 14.9 12.9 13.6		10,230 615,000 14,720,000 19.9 20.9 1.05 .605	
	270 200 135 19.6 19.3 31.5 23.5 15.5 15.2 24.9	55.6 .79 3.50 4.64 8.14 8.08 24.8	11,140_670,000 16,080,000 22.9 22.7 994 .526	
7:00 66.0 70 96 42 70 168 52.9 85.5	268 199 134 19.4 17.3 29.7 12.1 15.2 13.5 23.0		11,000 660,000 15,850,000 24.2 22.5 .93 .599	
10 54 95 41 70 172 52.9			11,000 660,000 15,853,000 21.9 22.0 1.03 .579	
	260 193 130 160 18.8 29.0 21.3 12.2 14.2 22.0	98.4     77       3.40     4.41       7.81     8.08       24.1     .54       3.28     4.41       7.69     8.08       20.6	9,130 555,000 13,309,000 17. 9 18.8 1.05 .520	Engine stopped 1:37
30 96 42 70 164 52.9	248 184 124 16.0 19.1 25.4 20.2 11.6 14.2 18.3		10,230 6/5,000 14,750,000 20.9	Intake suched dir till
8:30 68 54 82 28 70 180 53.0	292 216 146 20.7 20.3 29.2 23.4 17.7 17.3 26.0		10/200	9:00 P.M.
40 84 30 68 198 53.0	284 210 142 20.0 23.8 28.8 24.2 16.7 19.6 23.8	60.1 1.13 .67 .90 4.06 8.08 21.8	12,280 736,000 17,700,000 30.4 25.0 .825 695	2.00
50 86 32 68 195 52.9		60.3 1.35 1.04 1.195 4.06 5.75 9.81 8.08 273	12,750 766,000 18,384,000 30.7 26.0 .848 652	
9:00 63.0 68 87 33 67 204 52.9 120.	284 210 142 17.9 23.3 31.5 24.2 14.9 193 26.0		12,250 736,000 17,700,000 29.1 25.0 .86 605	
10 89 35 68 222 52.9	292 216 146 17.9 22.5 31.6 24.0 15.3 19.2 27.0		The state of the s	
10 88 34 68 216 52.9	290, 215 145 179 21.1 31.8 23.6 15.2 17.8 27.0		12,540 754,000 18,070,000 31.6 25.6 .85 .675	
30 53 88 35 68 210 516	288 213 144 22.0 21.1 32.0 25.0 18.5 19.7 26.9		12,800 768,000 18,950,000 31.4 26.1 1.21 625	
40 88 35 66 219 52.3	284 210 142 22.0 23.8 27.1 24.3 18.3 19.7 21.4		12,800 768,000 18,460,000 31.7 26.1 .823 .672	
50 87 34 66 207 523	284 210 142 22.4 22.7 34.6 26.5 18.5 18.8 28.6	65.9 1.35 1.04 1.195 4.52 5.20 9.72 8.08 28.3	12,800 768,000 18,450,000 31.4 26.1 .831 .612	
10:00 60.5 68 53 89 36 66 214 52.3, 154.	296 219 148 21.2 23.5 34.0 262 18.3 20.3 29.4	68.0 1.19 1.04 1.115 4.63 5.20 9.83 8.08 28.5	12,800 768,000 18,950,000 31.7 26.1 .822 .596	
10 89 36 67 202 52.2	190 215 145 21.2 22.5 32.1 252 17.9 19.0 27.2	64.1 1.30 1.04 1.170 4.63 5.25 9.88 8.08 28.5	12,800 768,000 18,450,000 31.8 26.1 .82 .637	
10 89 36 66 202 512	288 213 144 22.8 23.2 32.6 26.2 19.1 19.5 27.4	66.0 1.24 1.04 1.140 4.52 5.30 9.82 8.08 28.3	11 800 118,000 18,4571,000 31-7	*
30 53 88 35 67 208 522	284 210 142 22.8 24.1 34.6 27.1 18.9 19.9 28.6	67.4 1.24 1.08 1.160 452 3.20 9.72 8.08 23.1	13,032 784,000 18,800,000 32.3 28.0 .021	
40 88 35 67 212 52.2 116.5	284 210 142 21.2 23.3 33.0 25.8 17.8 19.3 27.3		12,800 768,000 18,450,000 31.1 26.1 .838 .610	
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11:00 61.0 68 53 88 35 66 210 522 42.5	280 207 140 21.6 23.2 26.5 23.8 17.7 19.0 20.8	575 107 1.02 1.045 4.07 5.10 9.17 808 28.2	12.680 160,000 18,230,000 30.2 26.0 .081	
11:00 61.0 68 53 88 35 66 210 522 112.5	295 218 147.5 26.7 23.8 28.6 26.4 23.1 21.3 24.7	49.1 1.13 1.06 1.495 9.57 5.15 9.67 8.08 78.7	try .	
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Rige 66.5 70.6 54.0 100-5 4.65 68.7 208.4 51.8 111.89	184 211 142 21.7 20.96 30.95 24.62 18.1 17.5 25.6	61.2 1.16 .928 1.044 4.23 4.83 9.06 0.08 21.1	[L,400 1.7,000 17,840,000 Lo. 0 Lo. 4	

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.50	07.5	65				273		609	302	283.	151	31.7 2	3.0 32	8 29.1	280	193	28.0	75.3	.80	79	795 4	1.87				11.150 670,000 161					
300 69	70		137			257		60.9	300	222	150	337 2	23.2			19.4	+		.825 .	.79	805 9	1.87				1,150 670000 161					
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·2+ ·32 68	70	65				350			278	222	150	33.0 2 29.1 2	7.0 74	4 30.	1 251	20.8	293									1590 695000 16					
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.48			131.5	_	-	276	60.8	60.0	276	207	138	33.4 2	26,2 29.4	4 29.	6 269	201	22.9	69.9	.675		13	5.10 3.8.	5 8.95	8.083							
56		66		63	77 5	280	60.9	60.9	304	223	152	30.8 2	64 32.	9 30.	0 27.2	224	28.2	77.8					7 8.97	8.083 2	3.8	10860 652000 15	550000 24.6		.908		
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./2								60.6	300	222	150	29.3 2	1.3 31.6	10 ZY.	1 25.7	230	26.2		.75	. 79	.776	1.53 3.8	5 8.38	8 083 2	5.00 /	11150 670000 16	100000 23.6	22.7	.962	432	
.28				75 ;		268				215	145	33.9 2	7.7 71.	0 30	7 28.6	223	254		.92	.96	746	4.76 3.4	8.16	8.083 2	7.40 1	2500 750000 /7	60000 25.7	25.5	.990	444	
.26		65				252		65	276	20\$	138	31.3 2	4.7 31.6	2 29.	1 25.3	19.0	24.3	68.6	.71	.875	797 3	3.96 3.7	4 7.70	8.083 2	5.75	11710 705000 16	900000 22.8	23.9	1.045	438	
.44				71.5		258 .		60	296	219	148	35.0 2	7.0 32.	0 31:	3 320	19.8	208	72.6	. 79	1.06	.930	4.82 3.9	7 8.79	8.083 2	8.4 1	2930 776000 18	70000 28.7	26.4	1 .920	+ .432	
.52		64.5	136	71.5	72 2	259	59.0	-	284	210	142	368 2	8.1 130.	4 31.8	30.0	22.4	24.3	76.7	.675	1.04	.857 4	1.63				12700 753000 18				506	-
5.00 65	- 72			69.5			7.0	-	290	2/5	143	30.2 2	3.5 29.	2 20.3	23.6	20.9	272	178 1	1.04	.00	1.010	413				12400 745060 17				.534	
.08				68. 7		250	58.7	615	300	272	150	29.4 2	85 27	7 28.	4 25.7	241	22.3	72./	1.21	1.12	1.160 4	186 4.1	9 9.05	8.083 2	9.7	13280 794 000 19	160000 33.3	27./		595	
.24		65		72 7				57	288	213	144	.72.7 2	7.0 70.7	2 29.8	3 27./	219	24.6	73.6		1.06	4	1.12 3.7	4 8.16	8.083 2	8.4	12430 776000 180	30000 27.7	26.4	1953	495	
.32				72.5 7				_	294	2/8	147	70.4 2	25.9 29.	9 28.7	7 28.0	215	24.8	74.3	1.27	1.12	1.190 4	176 3.9	7 8.73	8.083 2	9.7 /	13280 798000 19	150000 29.2	27./	.927	3/7	
.40		66	141.5	75.5 7	2 2	265	58.8	-	298	221	149	30.8 2	264 31.4	4 29.4	5 26.8	226	26.5	75.9	1.27 /	1.14	1.205 4	1.86 3.85	<b>5</b> 8.7/	8.083 3	0.0	13400 805000 19	310000 24.2	21.3	.927	5/2	
.50		-		77.5 7				=	304	223	152	32,4 2	5.6 29.	4 29.1	28.7_	220	25./	75.8	1.38	1.11	1.273 4	100 7.8	5 8.84	8.083 3	0.3	13520 813000 19 13400 805000 19	710000 29.8	27.7	. 930		
58 635		66.5		81.5 7				54	302	223	150	31.8 12	77 39	4 30.1	6 278	234	27.4	78.6 1	1.27	1.12	1.195 4	19.7 3.8.	5 8.79	8.083 2	9.7	13280 797000 19	150000 29.3	27.1		479	
6.04 60.		66.5		82.5 7				-	294	2./8	147	37.6 2	9.0 29.	4 32.	0 32.4	24.0	24.4	80.8	1.27	1.14	1.205	4.99 3.8	3 8.84	8.083 3	0.0	13400 805000 19	3/0000 29.7	21.3	. 9/6	487	
20		66.		73 7			500	-	282	209	141	7/.2 2	65 29	6 29.	1 26.8	217	244	72.9	1.15	1.16	1.155 4	1.51 3.7	4 8.25	7   8·083   2	28.8	12930 776000 18	630000 27.8	26.4	. 483	.502	C + + + + +
28		65.5		74.5			58.8	49.5	300	222	150	32.2	201 32	8 31	7 282	25.5	27.8	81.5	1,27	1.14	1.205 4	1.82 5.8	5 8.67	8.083 3	0.0	13400 805000 19	13/000 54.	3 61.0	./3	502	Generator thrown in.
36				77 7					292	2/6	146	32.0 2	.78 32	5 30.8	8 27.2	22.9	26.7	76.8	1.24	1.14	1.190 4	7.82 3.8	5 8.67	8.083	0.0	13400 805000 1	250000 27	7 25.9		323	
44		65	157	92 0		220		48	280	2/3	144	277 3	3.8 27.	0 27	7 227	204	275	67.8	1.10	1.10	1.100	152 7.7	4 8.24	8.083 3	0.5	12680 760000 18	000000 27.3	5 26.9		548	
52 7.00 50		64	10.6	88 4	7/	278	28.8 28.8	-	280	207	140	3/4 2	145 (14)	7) 23.	5 25.6	193	11.37	56.3	.93	.96	.945 4	23/ 3.74	4   8.05	8.083 2	1.5	12250 735000 17	670000 23.6	23.0	1.000		
108		11.	111-1	00	71	221	500		276	204	1.78	71.2 2	79 29	0 28.	0 25/	184	225	65.0 /	1.02	895	9575	4.31 3.83	3 8./6	8.083 2	6.2	11810 713000 11	100000 A71	7.4.6	1.7/0	- NAG	
.16		64	1455	81.5	78	220 :	58.8	-	250	185	125	23.8 2	7.9 30.	5 27.	4 17.4	160	21.5	549	.99	.855	.9225 :	3.97 3.5	2 7.44	8.083 2	7.6	11600 6960001	670 0000 El.	23.1	1.080	.527	
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-32	20-			75	70	216 .	58.8	-	264	195	132	728 2	8.5 74	4 30	6 268	224	240	7.72	.96	876	918	4.31 3.6	7 7.94	8.083.2	6.3	11710 705000 10	900000 23.5	23.9	1.015	1.423	
39	92,00	17	140	9/ .	71	221	688	_	286	202	143	31.7 3	02 3/0	3 31.	1 265	24.3	25.2	76.0	97	.42	.945	4.53 3.7	4 8.21	0.083 2	6.7	12000 120000 17	300000 23.0	24.3	.700	. 4 2 T	
.55		63	156	93	74	220	58.8		280	207	140	29.6 2	45 28	0 27.	3 24.2	/9.2	220	65.4	970	.42	.945	4.41 315	7.76	0.083 4	6.7	12000 120000 17	300000 64.0	E4.5	1.020	1114	T:
8.03	69.5	63	1585	95.5	78	238	588		284	200	142	3	33.0 29.	8	•	26.3	24.2	6	1475	.92	7475	430 3.6	2 7.43	8.083 2	67	12000 120000 11	300.000 ET!	AT.O	11:01-5		Fires cleaned
/0		63.	166	103	77	244	588	-	285	211	142.5		28.5 32					/	1.000	.98	.990	4.53 3.5	7 8.04	8.083 2	7.8	12400 745000 17	850000 25.1	25.0	1.010	1	
√8									280	20.7	140	Accession to the latest the lates	27.7 3/				24.8	/	196	875	9/75	4.00 3.5	7.93	8 087 2	4.1	1225073500017	900000 23.5	23.9	1.020		
.34				89						2/3		The state of the s	31.8 31.			25.8	30.2		1.04	.96	1:000	4.10 3.74	4 7.89	1 8 083 2	7.5	12250 735000 17	670000 24.3	25.0	1.030		:
8.42				133						2/9-			32.6 34.				285		1.06	.96	1.010	510 3.5	8.61	8.083 2	7.5	12250 735000 17	670000 261	25.0	.436	-	1004 81004 8.154
9.50			-				588.	_	288	2/3	144	1 5	76.7 29.	3		21.2	238		1.26	.7/	.985 4	1.76 3.6	3 8.39	8.083 2	4.9	11/50 670000 10	090000 23.5	238	.7/0		Jack Shaft Run by Auxiliary Engine.
9.55			-	-	-		58.5	-	288_	2/3	144	32.6	10.7 27	5 30	3 27.4	25.0	22,2	74.6	1.26	.67	.965	4.76 3.7	4 8.50	8.083 2	2.9	10250 615000 14	75000 220	220		393	noxing Figure
10-00				32			58.5	-	000	217	1111	20/	フファー ファ	6 70	2 219	22 -	27/	715	1.01	015	1977.5	11.5	4 1 14. 54	1 15.0X7 Z	6.7	11156 670000 10	300000 53.2	ET		447	
./0		6.5	/12	49		=	58.5	+	290	215	115	717	ファコーフ/	0 1 72	6 294	120	717	7/0	1911	102	.615	115 51	4 2.79	(   X.081   2	X-3	126XD /60000 '8	500000 EC.1		.970	489	
10.15		-		_	_	-	58.5	_	288	213.	144	33.0	22.1 30	4 28	5 27.7	17.8	24.7	702	1.15	1.08	1.115	4.65 3.7	4 8.39	8.083 2	19.2	13500 810000 18	800,000 28.5	27.6	.970	.534	
-				+==	-	17	-	-															,		1	634,950 38013000 91					,
TOTAL 5	15 42	5 3084	5 67763	37215	3328	11713	3085.7	1244.8	15360	0 2100	7680	1348.2	3387 140	07.1 115.	3.7 //3/.5	5 /07/3	1129.17	2848.6	51.095	47.951	47.9465 2	166 37	2/ 438.0	3 0 007	26/2	12450 746,00017	1910.000 26.6	265	.945	475	
AVGE	63.7 /(	64.	3 141.8	11.5	12.3	234.6	57.7	27.3	2420	214.0	14/5	32.1	21.5 30	1.0 29	6 21.0	21.9	24.1	13.0	704	.740	702	4.00 3.7	8.4	2 8.085	20.0	12,450 746,00017	,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,		1		

OBJECT STEST-Mechanical Efficiency of Engine BEROMETER- 28.35 WEATHER CONDITIONS- Fair.

		MEDI	8 21 18
	7	MECH	REMARKS
73	TOTAL	EFF	
	TOTAL 20	2/	
1.4	93.0	84.6	
4.7	99.0		
29.0	94.3	The second leaves of the second	
35.0	114.8	77.5	722 8 4 5 4 4 2 10 10
0.40			500 P.M Spark advanced to 180°,
24.2	QA F	170	
28.0	90.5	47.0	
17.0	86.7	49.5	
2 <i>5.0</i> 33.9	8 <i>9.5</i> 93.6	79.5	
34.5	93.2	79.5	
10.4 35.0	114.0	92.1	
טיענ	103.0	76.1	
43.9	1061.6	864.5	
31.2	96.6	78.6	

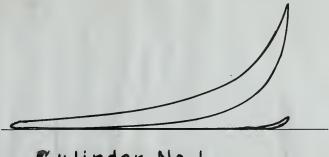
2 4.37 1 hr. 3 min.

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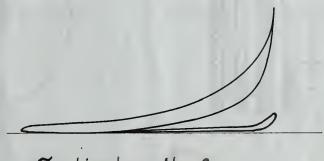
181 CT. T- Mechanical Efficiency of Engine

0' ETE 28.35
V - ATHER COVDI ONS Fair.

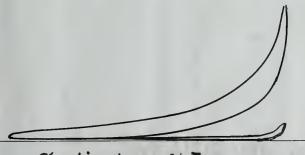
## CASTUATER 508   234   256   128 525 786
## 58   73   80   234   256   28   325   786   406   363   425   418   340   270   314   930   846    ## 58   37   79   234   236   140   560   920   427   370   336   384   350   303   290   943   975    ## 58   37   79   234   280   140   560   920   427   370   336   384   350   303   290   943   975    ## 59   57   336   79   231   340   170   (75)   165    ## 50   60   1/5   55   340   170   (75)   165    ## 20   665   120   575   340   370   380   380    ## 24   58   60   1/0   50   256   128   48   356   48   350   373   38   378   379    ## 30   60   1/0   50   256   128   48   356   470   370   372   374   375   380    ## 31   575   576   576   576   576    ## 32   576   576   576   576   576   576    ## 32   576   576   576   576   576    ## 32   576   576   576   576   576    ## 32   576   576   576   576    ## 32   576   576   576   576   576    ## 32   576   576   576   576   576    ## 32   576   576   576   576   576    ## 32   576   576   576   576   576    ## 32   576   576   576   576   576    ## 32   576   576   576   576   576    ## 32   576   576   576   576   576   576    ## 32   576   576   576   576   576   576   576   576    ## 32   576   576   576   576   576   576   576   576   576   576    ## 32   576   576   576   576   576   576   576   577   577   576   576   577   577   576   576   577
## 58   78   80   234   256   128   525   786   446   363   425   478   346   270   374   930   846   347   580   137   79   231   308   154   525   350   473   337   387   377   3
## 58   73   80   234   256   728   525   786   406   363   425   478   346   270   314   930   846   ## 58   73   79   231   308   545   525   930   4/3   337   387   377   226   347   990   900   ## 58   73   79   234   280   40   560   920   427   370   356   384   350   303   290   943   975   ## 9 57   736   79   231   316   58   480   890   488   396   381   415   433   365   350   1148   775   ## 9 57   736   79   231   340   707   759   105   ## 10 60   1/5   55   316   158   225   380   225   235   ## 226   605   120   395   370   155   235   426   316   328   328   328   328   905   470   ## 128   60   1/0   50   226   128   428   356   410   386   377   332   333   272   280   905   470   ## 128   60   1/0   50   226   128   425   40   400   347   335   351   297   28   256   805   795   ## 130   60   1/0   50   2290   145   130   730   117   279   400   372   344   253   339   936   780   ## 131   60   122   620   222   146   555   936   473   373   374   243   345   932   785   ## 131   60   122   620   222   146   555   936   473   373   474   476
140 58 37 79 231 308' 154 525 525 526 413 337 387 379 417 226 347 990 943 975 149 557 736 560 820 427 370 356 384 350 303 290 943 975 149 57 736 79 231 316 158 480 890 468 396 384 415 433 363 350 1/48 775 340 770 (75) 163 340 170 170 (75) 163 340 170 170 (75) 163 340 170 (75) 170 (75) 170 (75) 170 (75) 170 (75) 170 (75) 170 (75) 170 (75) 170 (75) 170 (75) 170 (75) 170 (75) 170
## 58   37   79   234   280   140   560   920   427   370   356   384   350   303   290   943   975   316   158   480   890   468   396   381   415   433   365   350   114.8   775   350   340   170   1757   165   340   170   1757   165   350   340   170   1757   355   340   158   205   380   383   299   330   333   292   280   905   470   320   3
# 57   36   79   23    316   58   480   890   468   396   381   415   433   365   350   1/48   775    # 50   340   170   175   181   182    # 510   60   1/5   55   376   158   205   380    # 120   605   120   395   310   155   235   426   368   328   297   33.0   33.3   292   280   905   470    # 128   60   1/0   50   256   128   425   418   356   410   486   317   328   388   70   867   495    # 130   60   1/0   50   256   128   425   400   400   547   33.5   361   297   258   250   895   795    # 130   60   1/0   50   290   143   430   710   417   299   400   372   344   253   339   936   785    # 137   60   122   620   292   146   555   950   475   373   474   446   419   317   404   1/40   833    # 150   595   144   845   274   137   590   950   479   371   436   426   383   297   350   1030   821    # 150   596   145   362   2325   296   148   4329   732   428   352   372   384   363   291   312   966   786    # 150   596   145   360   125   2325   296   148   4329   732   428   352   372   384   363   291   312   966   786    # 150   596   146   565   577   7285   930   3842   792   5195   8782   4705   3879   4097   4221   3994   3202   3439   1616   8645    # 150   596   146   475   57
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20   605   120   595   310   155   235   426   368   323   297   330   33.3   292   280   90.5   47.0     24
316   158   225   418   35.6   41.0   186   31.7   32.8   37.8   17.0   86.7   49.5     28
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33   595   1/0   50.5   288   144   440   742   418   29.0   41.1   37.3   34.4   24.3   34.5   93.2   79.5
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3 540. 595   44 84.5   274   37 590   9.50   47.9   37.1   43.6   42.6   383   29.7   35.0   103.0   92.1    Toral   650.5   379   728.5   930   3842   7921   519.5   878.2   470.5   387.9   4097   4221   3994   320.2   343.9   1061.6   864.5    ###
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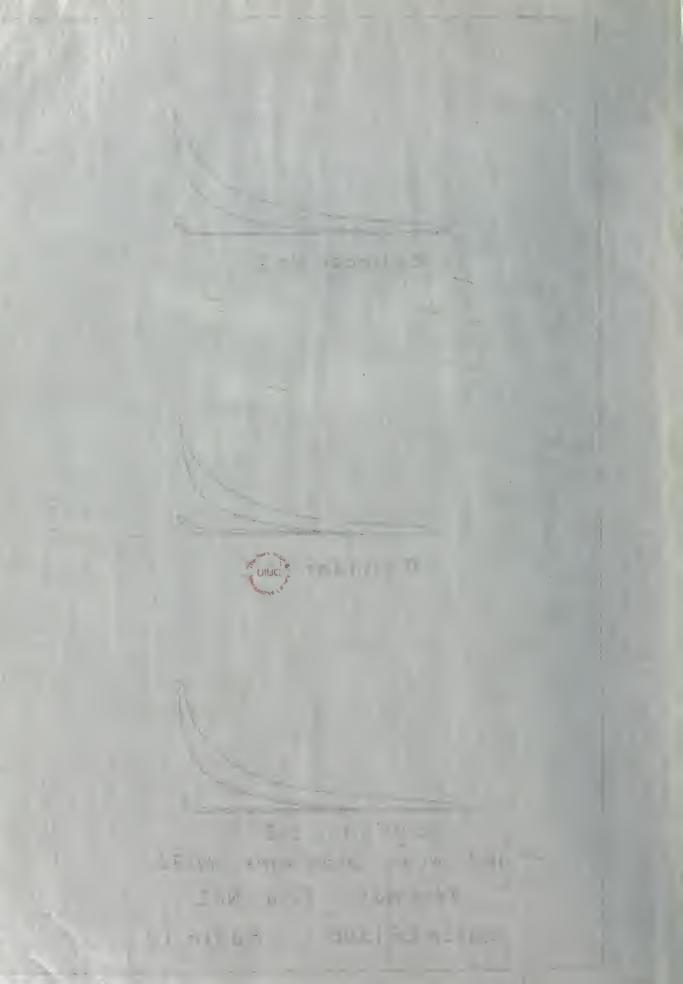
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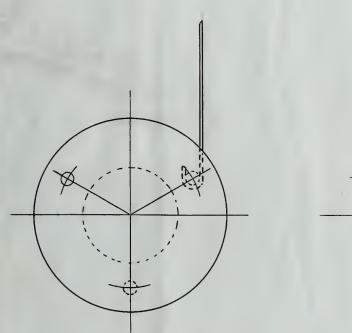


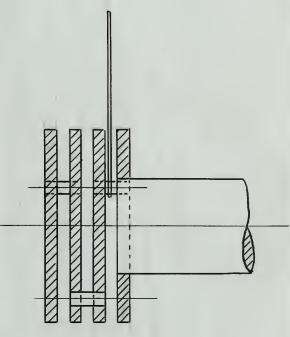
Cylinder No. 2



Cylinder No.3
Indicator Diagrams No.33
Test No.1, Trial No.2
March 26,1908 Pekin, 111.







REDUCING MOTION

